# **Deflection of Beams** Lecture 3 - Alternative Loading Types

Department of Mechanical, Materials & Manufacturing Engineering **MMME2053 – Mechanics of Solids**



## **Deflection of Beams**

#### **Learning Outcomes**

- 1. Know how to derive the differential equation of the elastic line (i.e. deflection curve) of a beam (synthesis);
- 2. Employ Macaulay's method, also called the method of singularities, to determine bending moment expressions for beams where there are discontinuities in the bending moment distribution arising from discontinuous loading (application);
- 3. Be able to solve this equation by successive integration in order to yield the slope,  $\frac{dy}{dx}$ , and the deflection, y, of a beam at any position,  $x$ , along its span (application);
- 4. Recognise and use different singularity functions in the bending moment expression, relating to different loading conditions, including point loads, uniformly distributed loads and point bending moments (comprehension);
- 5. Employ Macaulay's method for statically indeterminate beam problems (application).

## **Alternative Loading Types**

#### **Uniformly Distributed Load**

Consider a uniformly distributed load (UDL),  $w$  Nm<sup>-1</sup>, acting over part of a beam's span.



The UDL runs from distance  $a$  from the origin (left-hand end) of the beam, all the way to the right-hand end of the beam. A discontinuity occurs at the position where the UDL commences.

Drawing a free body diagram of the beam sectioned after the discontinuity:



Taking moments about the section position in order to determine an expression for the bending moment,  $M$ :

$$
M = R_A x - \frac{w\langle x - a \rangle^2}{2}
$$

As can be seen, when taking moment equilibrium, the contribution of the UDL is calculated by first turning the UDL,  $w$ (unit Nm<sup>-1</sup>), into a force (unit N) by multiplying it by the distance over which it acts,  $x - a$  (unit m). This force is then turned into a moment by multiplying it by the distance to the centre position of the UDL,  $\frac{\langle x-a\rangle}{2}$  (unit m).

#### **Discontinuous Uniformly Distributed Load**

Consider a discontinuous uniformly distributed load (UDL),  $q$  Nm<sup>-1</sup>, acting over part of a beam's span.



In this case, the UDL, q, runs from distance a from the origin (left-hand end) of the beam, up to distance b from the origin. Discontinuities therefore occur both at the position where the UDL commences, and at the position where it ends.

In order to progress towards a general bending moment expression for the beam, the applied discontinuous UDL,  $q$ , is extended to the end of the beam and an additional, negative, counterbalancing UDL superimposed over the newly extended part.



The extended applied UDL, q, and the added counterbalancing UDL,  $-q$ , mathematically cancel each other out and therefore this gives a statically equivalent system to the original partially extended (discontinuous) UDL.

Drawing a free body diagram of the beam sectioned after the discontinuity:



Taking moments about the section position in order to determine an expression for the bending moment,  $M$ :

$$
M = R_A x + \frac{q\langle x-b\rangle^2}{2} - \frac{q\langle x-a\rangle^2}{2}
$$

#### **Point Bending Moment**

Consider a point bending moment,  $M_o$  Nm, acting at a distance  $a$  from the left-hand side of a beam.



This point bending moment gives rise to a discontinuity in the bending moment expression.

Drawing a free body diagram of the beam sectioned after the discontinuity:



Taking moments about the section position in order to determine an expression for the bending moment,  $M$ :

$$
M = R_A x - M_o \langle x - a \rangle^0
$$

Note that the form of the discontinuity function for the point bending moment is the same as for a point load, except that the bracketed length is raised to the power zero. This is simply a mathematical convenience for facilitating Macaulay's method whilst maintaining the correct units of the moment,  $M_a$ . I.e. if a position in the beam where  $x < a$  is considered for evaluation, then the contents of the Macaulay brackets is negative and the entire term set to zero. However, if a position in the beam where  $x > a$  is considered for evaluation, then the contents of the Macaulay brackets is positive, and the term is included – but as the length term,  $(x - a)^0$ , is raised to the power of zero, it becomes 1, and so the term simplifies to  $M_o$ .

## **Summary of the Discontinuity Functions**

When developing the bending moment expression for a beam with load discontinuities, the singularity functions used for each different type of load are:



Note that in these singularity functions, the exponent is 1 for a point load, 2 for a UDL and 0 for a point bending moment.

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